

# Introduction

1

## Measurement planning

- Some everyday examples - Temperature outside
- Weight of the body

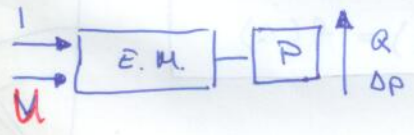
indirect quant. accuracy

1 device → immediately obtained results  
Why is it important measurement planning?

- In engineering the situation is more complicated!

## Example 1

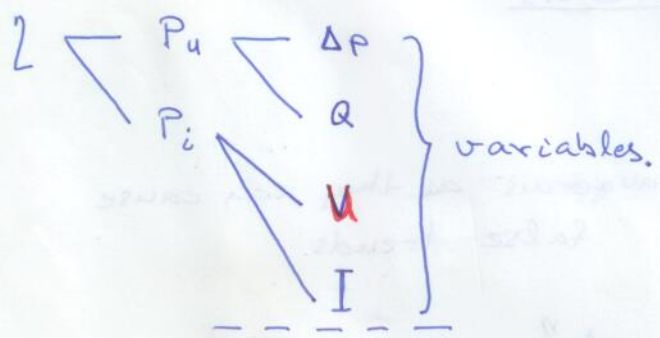
Measurement of the efficiency of an electric motor-pump group



Efficiency ( $\eta$ ) ?

there is no device to measure directly!

$$\eta = \frac{\text{useful } P_u}{\text{input } P_{in}} = \frac{\Delta P \cdot Q}{I \cdot U}$$



Determine the required variables and the possible parameters

- $n$  - revolution number
- $T$  - liquid temperature,  $\mu = f(T)$  viscosity.
- $f_s$  - electric network frequency  $\approx 50 \text{ Hz}$
- ...
- ...

Parameters

$$\eta = f(\Delta P, Q, U, I; n, T, f_s \dots)$$

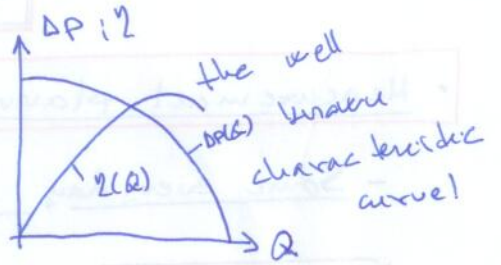
# Dependent / Independent variables

$$\Delta P = f(Q)$$

$$Q = f(\Delta P)$$

$V$  - constant, independent

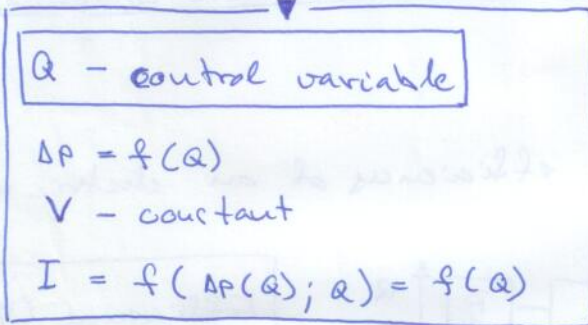
$$I = f(\Delta P, Q)$$



How many ~~independent parameters~~ variables can describe the state of the system?

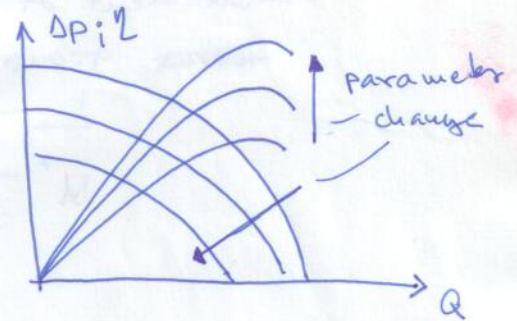
- Only 1, but 2 possible choice:  $Q$  or  $\Delta P$

Let us choose  $Q$



extraneous

Determine the control variables



All the other variables are extraneous!  
(fixed)

- We cannot or do not want to control.

## Controlled / extraneous parameters

$n$  - controlled

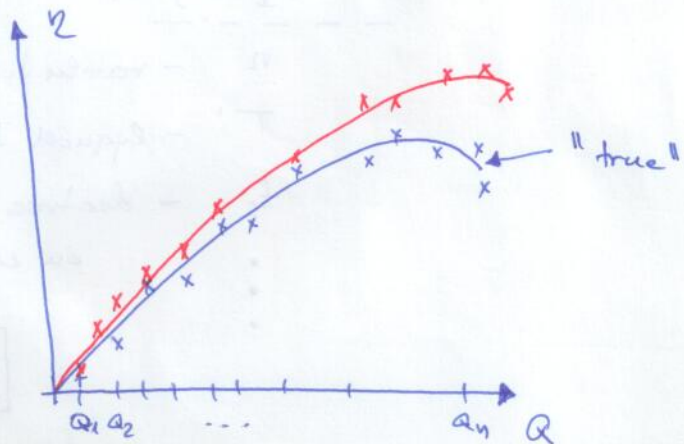
$T$   
 $f_s$  } extraneous → Dangerous as they can cause false trends.

for instance

•  $Q$  changed continuously.

•  $T$  changed

Interference  
(false trend)



## Solution

- Make it controllable

e.g. cooling device

- Random measurement

Originally:  $Q$  was increased from  $Q_{min}$  to  $Q_{max}$  continuously!  $(Q_1, Q_2, \dots, Q_n)$

Random:  $(Q_{s1}, Q_{s2}, Q_{s1}, Q_{s1}, Q_{s1}, \dots, Q_{sn})$

↓  
break the trend

↓  
became random error like noise

↓  
handled by statistics

## Statistics

- Every measured variable contain random variation (noise) or other kind of uncertainty.
- The true value of the desired variable  $X$  is not known.
- Estimation for  $X$  is:

$$X \approx X' = \bar{X} \pm u_x \quad (P\%)$$

estimation ←

→ confidence e.g. 95%

→ uncertainty

→ average

- For dependent variables (for instance the efficiency)  
(comp/α)

$$y = f(x_1, x_2, \dots, x_n)$$

↓  
 $u_{x_1}$

↓  
 $u_{x_2}$

↓  
 $u_{x_n}$

→  $u_y = ?$

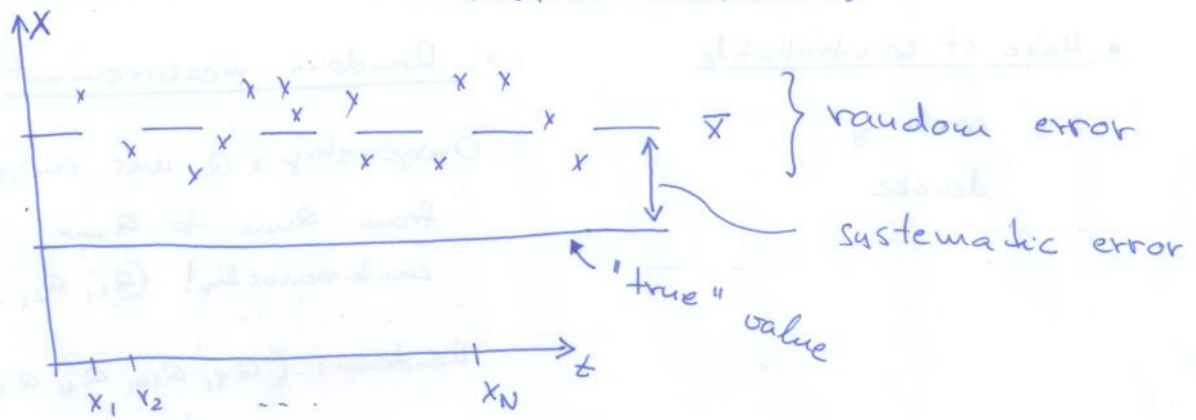
Error propagation!

Sensitivity analysis!

↓  
Choice of measurement device!

uncertainty of a single variable

(direct measurable)



- random error (confidence interval)

$$U_r = t_{st} \cdot \frac{s^*}{\sqrt{N}}$$

$t_{st}$  - Student coefficient

$$t_{st} = f(P; N-1)$$

degree of freedom

confidence/probability

$s^*$  - standard deviation

- systematic error

- calibration →  $U_s = \phi$

- if calibration is not possible:

data sheet: Pressure transducer

Range: 1-10 bar ( $P_{max} = 10 \text{ bar}$ )

Class of accuracy:  $E_{cal} = 0,5\%$  - usually in terms of  $P_{max}$

$$U_s = P_{max} \cdot E_{cal} = 10 \text{ bar} \cdot 0,005 = \underline{0,05 \text{ bar}} \quad (X_{max})$$

$$U_x = \sqrt{U_r^2 + U_s^2}$$

- other errors similarly:

Linearity:  $E_{lin}$

Hysteresis:  $E_{hys}$

⋮

$$U_x = \sqrt{U_r^2 + U_s^2 + U_{lin}^2 + U_{hys}^2 + \dots}$$

uncertainty of indirect variables

$$y = f(x_1, x_2, \dots, x_n)$$

Error propagation:

$$u_y^2 = \left(\frac{\partial f}{\partial x_1}\right)^2 \cdot u_{x_1}^2 + \left(\frac{\partial f}{\partial x_2}\right)^2 \cdot u_{x_2}^2 + \dots + \left(\frac{\partial f}{\partial x_n}\right)^2 \cdot u_{x_n}^2$$

$\frac{\partial f}{\partial x_i}$  - sensitivity coefficient!

if  $\partial f / \partial x_i$  big  $\rightarrow$  small error will be magnified!

$\downarrow$   
more precise instrument is needed!

Example 2

Density measurement of a cylinder

$$\rho = \frac{m}{V}, \quad V = \frac{d^2 \pi \cdot H}{4}$$

$$\rho = \frac{4 \cdot m}{d^2 \pi H}$$

Variables, Data Sheet

- $E_m = 0.5\%$ ;  $m_{max} = 5 \text{ kg}$
- $E_d = 0.5\%$ ;  $d_{max} = 500 \text{ mm}$
- $E_H = 0.5\%$ ;  $H_{max} = 500 \text{ mm}$

• measurement results

- $U = 10$  for each variable
- $\bar{m} = 2000 \text{ g}$ ;  $S_m^* = 18 \text{ g}$
- $\bar{d} = 101.5 \text{ mm}$ ;  $S_d^* = 0.51 \text{ mm}$
- $\bar{H} = 402 \text{ mm}$ ;  $S_H^* = 1.3 \text{ mm}$
- $P = 95\%$

$$t_{95} = f(n-1; P) = \underline{\underline{2.123}}$$

$$\left. \begin{array}{l} u_r^m = 12.7 \text{ g} \\ u_s^m = 25 \text{ g} \end{array} \right\} u_m = \underline{\underline{25 \text{ g}}}$$

$$\left. \begin{array}{l} u_r^d = 0.36 \text{ mm} \\ u_s^d = 2.5 \text{ mm} \end{array} \right\} u_d = \underline{\underline{2.52 \text{ mm}}}$$

$$\left. \begin{array}{l} u_r^H = 0.92 \text{ mm} \\ u_s^H = 2.5 \text{ mm} \end{array} \right\} u_H = \underline{\underline{2.66 \text{ mm}}}$$

$$\frac{\partial \rho}{\partial m} = \frac{4}{d^2 \pi H} \cdot \frac{\partial \rho}{\partial d} = -\frac{8m}{d^3 \pi H} \cdot \frac{\partial \rho}{\partial H} = -\frac{4m}{d^2 \pi H^2}$$

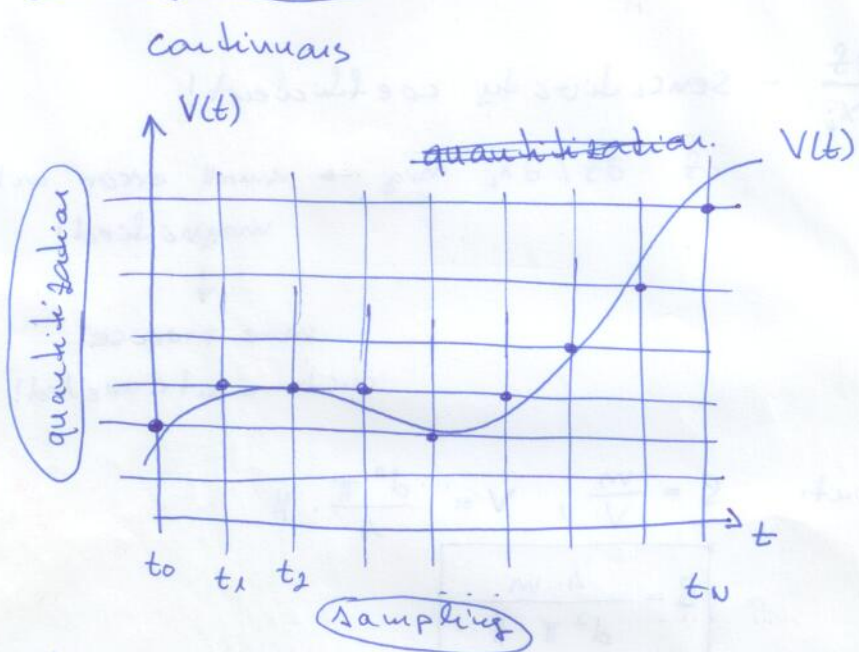
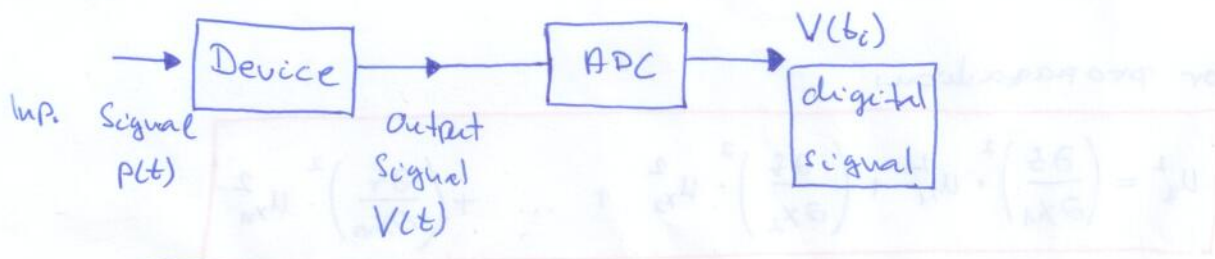
$$\begin{aligned} \left(\frac{u_\rho}{\rho}\right)^2 &= \left(\frac{u_m}{m}\right)^2 + \left(\frac{2 \cdot u_d}{d}\right)^2 + \left(\frac{u_H}{H}\right)^2 = \\ &= 8.77 \cdot 10^{-5} + \underline{\underline{246.6 \cdot 10^{-5}}} + 4.38 \cdot 10^{-5} = \\ &= 259.7 \cdot 10^{-5} \end{aligned}$$

$$\frac{u_\rho}{\rho} = 0.05096 \rightarrow \frac{u_\rho}{\rho} \approx \underline{\underline{5.1\%}}$$

# Data Acquisition

2

• Problem



$N$  - number of measurement

Questions: - How to choose the sample rate?

- What will be the resolution of signal  $V(t)$ ?

↓

during quantization.

How can we extract the important feature of  $V(t)$

10p

• Quantization error

• Digital devices → binary code

Unit: bit 0 or 1

Series of bit: word e.g. 1001010001 16-bit word.

Specially: 8 bit word = byte

↓

8 bit system has	$2^8$	= 256	different words
16	-	$2^{16}$	= 65,536
32	-	$2^{32}$	= 4,294,967,296

Generally

An  $M$ -bit system has  $2^M$  different words!

The full scale range  $U_{FSR} = U_{max} - U_{min}$  is divided into  $Q = \frac{U_{FSR}}{2^M - 1}$  intervals.

So the resolution:

$$Q = \frac{U_{FSR}}{2^M - 1} = \text{LSB}$$

least significant bit

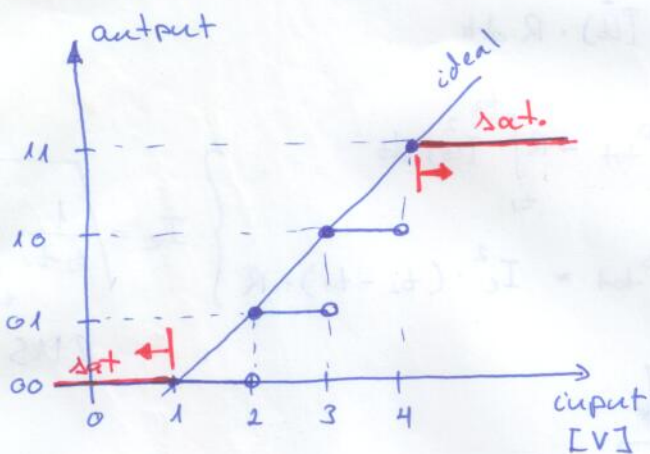
this is the least  $V$  change, which cause 1 bit change in the output!

Sp

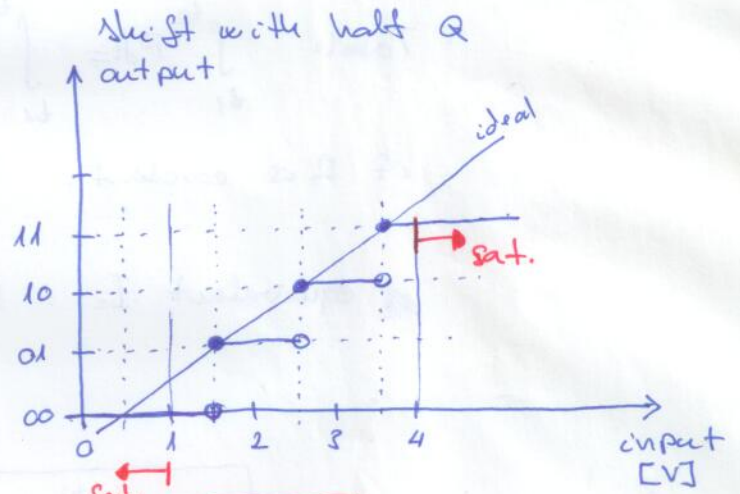
Example 3

Pressure tr. 1-4 V output  
2 bit ADC

$$Q = \frac{4-1}{2^2-1} = \frac{3}{3} = 1 \text{ V}; 2^2 = 4 \text{ words}$$



$$U_q < Q$$



$$U_q \pm \frac{Q}{2}$$

10p

Example 4

$U$  range: 0-10V  
ADC: 8 bit

$$Q = \frac{U_{FSR}}{2^8 - 1} = \frac{10V}{255} = \underline{\underline{39.2 \text{ mV}}}$$

$U_i = 100 \text{ mV}$

$$U_q = \pm \frac{Q}{2} = \underline{\underline{19.6 \text{ mV}}}$$

Sp

Relative Error,  $\frac{U_q}{U_i} = \frac{19.6 \text{ mV}}{100 \text{ mV}} = \underline{\underline{19.6\%}}$  BIG!!!

- Solutions:
- Increase the number of bits!
  - Decrease  $U_{FSR}$ ? → smaller measurement range!

$p \neq mbar$ ; do not use:  $P$  range 0-10 bar!!

# System Behaviour

5

• The measurement device is a dynamical system.

↳ Example: measurement of body temperature!  
There is a response time.

• So far: Static calibration (in principle fast response)

• Dynamic calibration (Response characteristics for different input signals)

## System model

• Usually linearity is important during measurement

↳ usually the system can be modelled with linear models

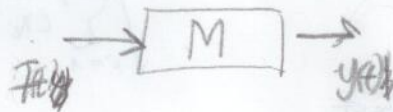
• In general:

$$a_n \frac{dy^n}{dt} + a_{n-1} \frac{dy^{n-1}}{dt} + \dots + a_1 \frac{dy}{dt} + a_0 y = F(u)$$

$a_i$  - system coefficients.

↑  
input / "true" signal

## 3 cases



• Zero-order System (instant response time)

• First-order (capacity)

• Second-order (moment of inertia)

} - Step function.  
- sine

$$T_d \cdot \dot{y} = T$$



**Zero - order system**

$a_0 y = F(t)$

$y = k F(t); k = \frac{1}{a_0}$

- static sensitivity

slope of the static calibration curve!

$y = k F$

- Indicial response time.
- For example: static calibration.

e.g. pressure transd.  
 $y [V]$   
 $F [bar]$   
 $k [V/bar]$

**First - order system**

$a_1 \frac{dy}{dt} + a_0 y = F(t)$

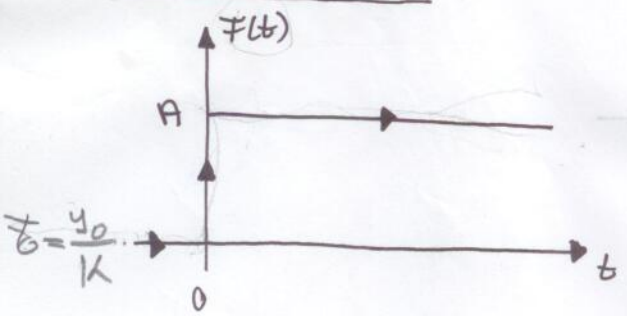
capacitive storage elements

$\tau \frac{dy}{dt} + y = k F(t)$

$\tau = \frac{1}{a_1}; \tau = \frac{dQ}{dt}$

$\tau$  - time constant  
 $k$  - static sensitivity

Step function:



$\tau \cdot \dot{y} + y = k \cdot A$   
 $y(0) = y_0$

Solution is the exponential function.

$y_H = C \cdot e^{\lambda t}; \dot{y}_H = \lambda \cdot C \cdot e^{\lambda t}$

Homog:  $\tau \lambda C e^{\lambda t} + C e^{\lambda t} = 0$  - characteristic eq.  
 $\tau \lambda + 1 = 0$   
 $\lambda = -\frac{1}{\tau} \rightarrow y_H = C \cdot e^{-\frac{1}{\tau} t}$  ( $\tau \dot{y} + y = 0$ )

Inhom: (Similar form than the differential)  
 polynomial  $\rightarrow$  polynomial  
 trigonometric  $\rightarrow$  trigonometric.

$\dot{x} = x$

$$y_{IH} = C_{IH} \cdot$$

$$C_{IH} = ?$$

$$\dot{y}_{IH} = \phi$$

Substitute into the equation,

$$C_{IH} = kA$$

$$y = y_H + y_{IH}$$

$$y = kA + C \cdot e^{-\frac{1}{\tau} t} \quad \text{— general solution}$$

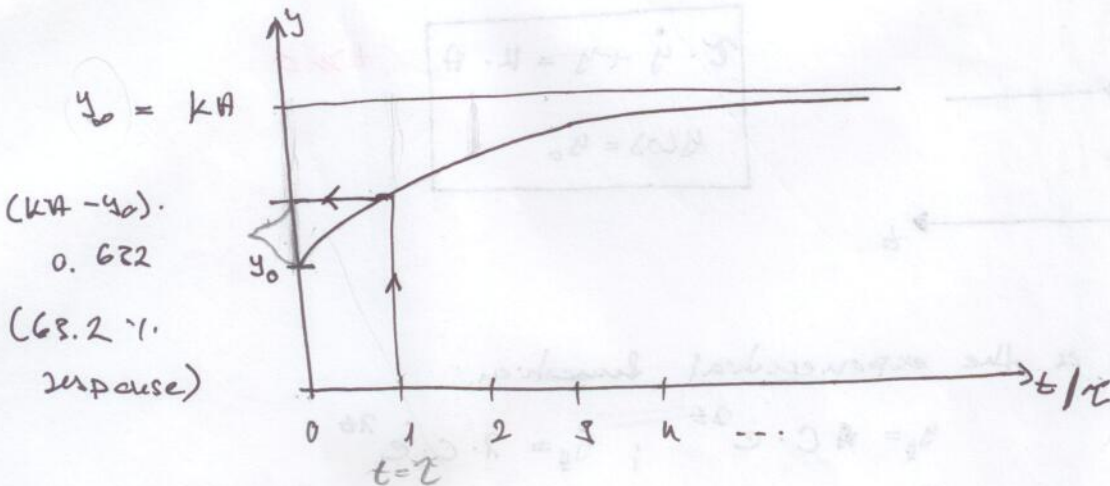
I.C.  $y(0) = y_0$

$$y_0 = kA + C \cdot \frac{e^{-\frac{1}{\tau} \cdot 0}}{1} \rightarrow C = y_0 - kA$$

$$y(t) = kA + (y_0 - kA) \cdot e^{-\frac{1}{\tau} t}$$

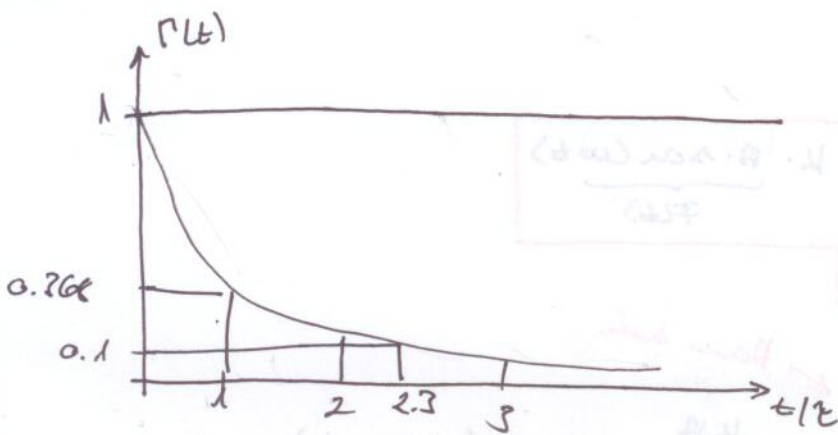
Steady response

transient response



Error function:

$$\Gamma(t) = \frac{y(t) - y_0}{y_0 - y_0} = e^{-\frac{1}{\tau} t}$$



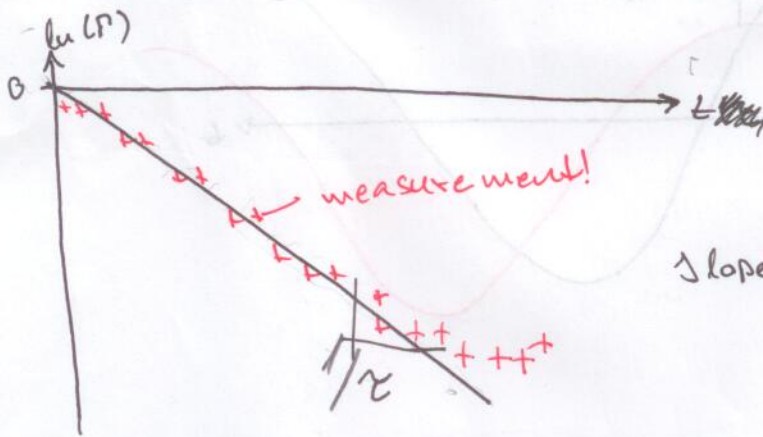
$t/\tau = 1$	$\rightarrow \Gamma(t) = 0.368 = 36.8\%$	$\rightarrow \tau$ is how fast the system responds.
$= 2$	$0.135 = 13.5\%$	
$= 3$	$0.05 = 5\%$	$\tau$ is the time needed to achieve 63.2% response.
$\approx 2.3$	$0.1 = 10\%$	

$\rightarrow$  rise time of the system  $\rightarrow \tau_r = 2.3 \cdot \tau$

$$\Gamma(t) = e^{-t/\tau}$$

$$\ln(\Gamma)$$

$\ln(\Gamma) = -\frac{1}{\tau} \cdot t \rightarrow$  linear on a semi-log plot



Slope of the curve is the time constant



we can, if the system

is not linear  $\rightarrow$  the measured points will deviate from the curve!

• Sine function input

$$\tau \cdot \dot{y} + y = k \cdot \underbrace{A \cdot \sin(\omega t)}_{F(t)}$$

I.C:  $y(0) = y_0$

~~General~~  
~~Specific~~ solution:

$$y(t) = C \cdot e^{-\frac{t}{\tau}} + \frac{kA}{\sqrt{1+(\omega\tau)^2}} \cdot \sin(\omega t - \phi)$$

transient

steady state

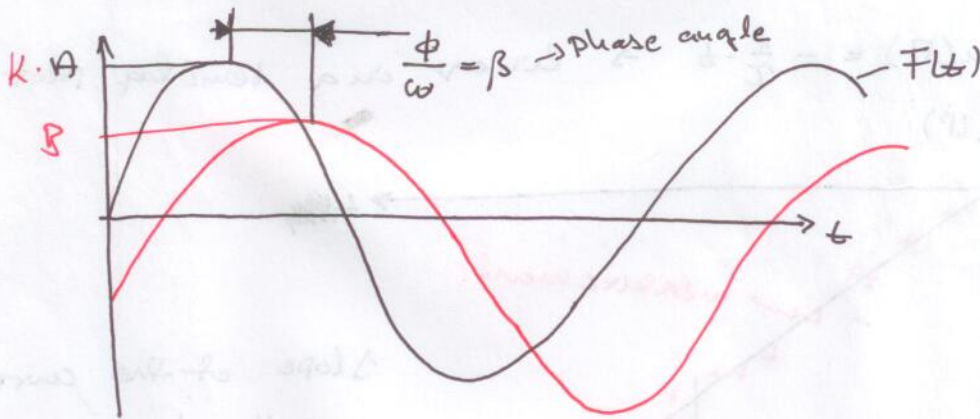
Hand. sol.

Long-term behaviour:

$$y_{ss}(t) = B(\omega) \sin(\omega t - \phi(\omega)) = B(\omega) \sin\left(\left(t - \frac{\phi}{\omega}\right)\omega\right)$$

$$B(\omega) = \frac{kA}{\sqrt{1+(\omega\tau)^2}} \quad \text{— amplitude of the steady response}$$

$$\phi(\omega) = \tan^{-1}(\omega\tau) \quad \text{— phase shift}$$



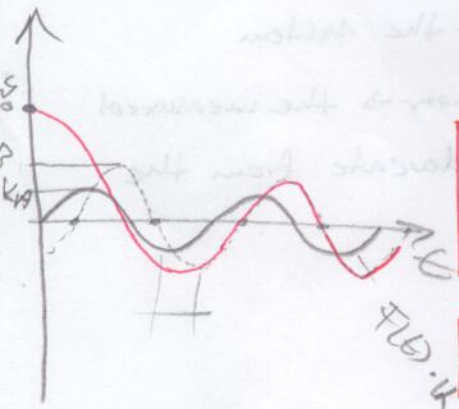
• Amplification:

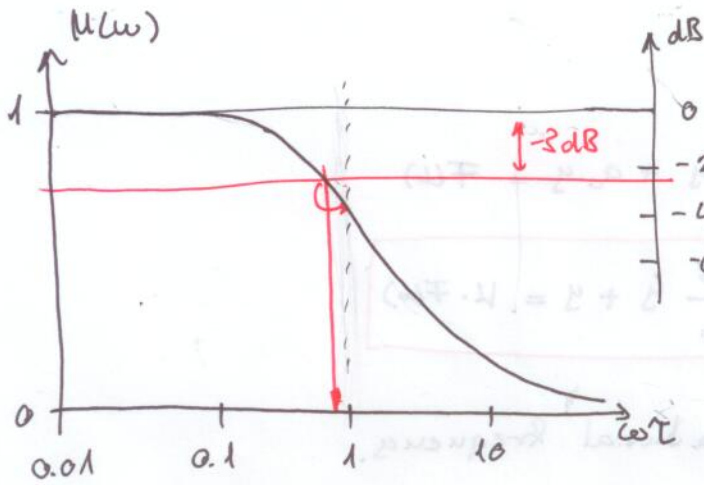
$$M(\omega) = \frac{B}{k \cdot A} \quad \rightarrow \text{ampl. response in [V]}$$

$\rightarrow$  ampe of excitation also in [V]

$$M(\omega) = \frac{1}{\sqrt{1+(\omega\tau)^2}}$$

$$\phi(\omega) = \tan^{-1}(\omega\tau)$$





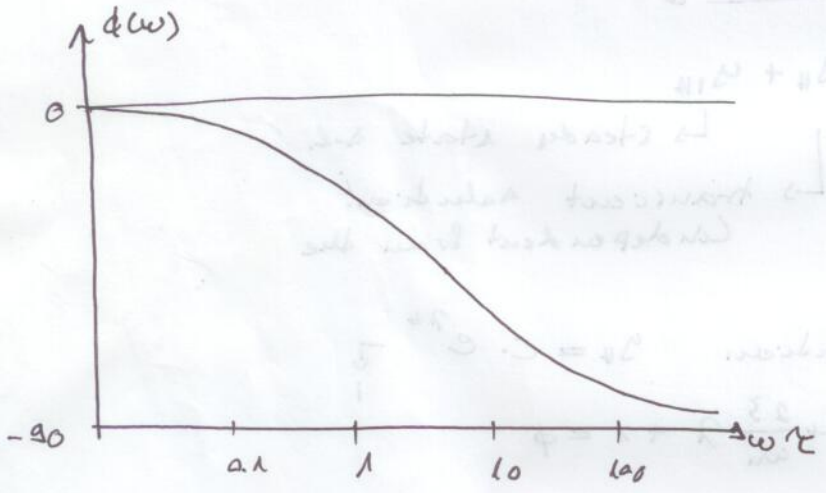
instans at 0 - bross?

$M(\omega) \ll$  such frequencies components are filtered!

$M(\omega)$  in dB =  $20 \log M(\omega)$

Frequency Bandwidth

$M(\omega) \geq -3 \text{ dB}$   
 $M(\omega) \geq 0.707 \rightarrow \omega_c T = 1$



• Phase linearity is important, usually a signal contains multiple frequencies!



• Discussion on Second-order systems!

• Second-Order system

$$a_2 \ddot{y} + a_1 \dot{y} + a_0 y = F(t)$$

$$\frac{1}{\omega_n^2} \ddot{y} + \frac{2\zeta}{\omega_n} \dot{y} + y = k \cdot F(t)$$

$$\omega_n = \sqrt{\frac{a_0}{a_2}} \quad - \text{natural frequency.}$$

$$\zeta = \frac{a_1}{2\sqrt{a_0 a_2}} \quad - \text{damping ratio.}$$

$$k = \frac{1}{a_0} \quad - \text{static sensitivity.}$$

$$I.C. \quad y(0) = y_0; \quad \dot{y}(0) = \dot{y}_0$$

Solution:

$$y(t) = y_H + y_{IH}$$

↳ steady state sol.  
 ↳ transient solution.  
 Independent from the

• Homogeneous sol.

characteristic equation  $y_H = C \cdot e^{\lambda t}$

$$\frac{1}{\omega_n^2} \lambda^2 + \frac{2\zeta}{\omega_n} \lambda + 1 = 0$$

$$\lambda_{1,2} = -\zeta \omega_n \pm \omega_n \sqrt{\zeta^2 - 1}$$

a)  $0 \leq \zeta < 1$  (under damped system)

$$y_H(t) = C \cdot e^{-\zeta \omega_n t} \sin(\omega_d t + \phi)$$

oscillatory behaviour

b)  $\zeta > 1$  (overdamped system)

$$y_H(t) = C_1 e^{\lambda_1 t} + C_2 e^{\lambda_2 t}$$

smooth response

c)  $\zeta = 1$  (critically damped system)

$$y_H(t) = C_1 e^{\lambda t} + C_2 t e^{\lambda t} \quad \text{divide } a_1 \text{ and } b_1$$

• Step function

$F(s) = A \rightarrow Y_0 = K \cdot A$

large bandwidth system

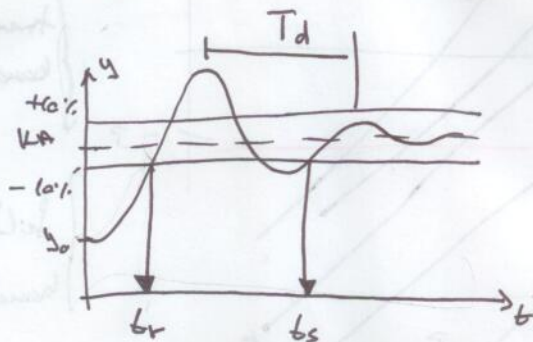
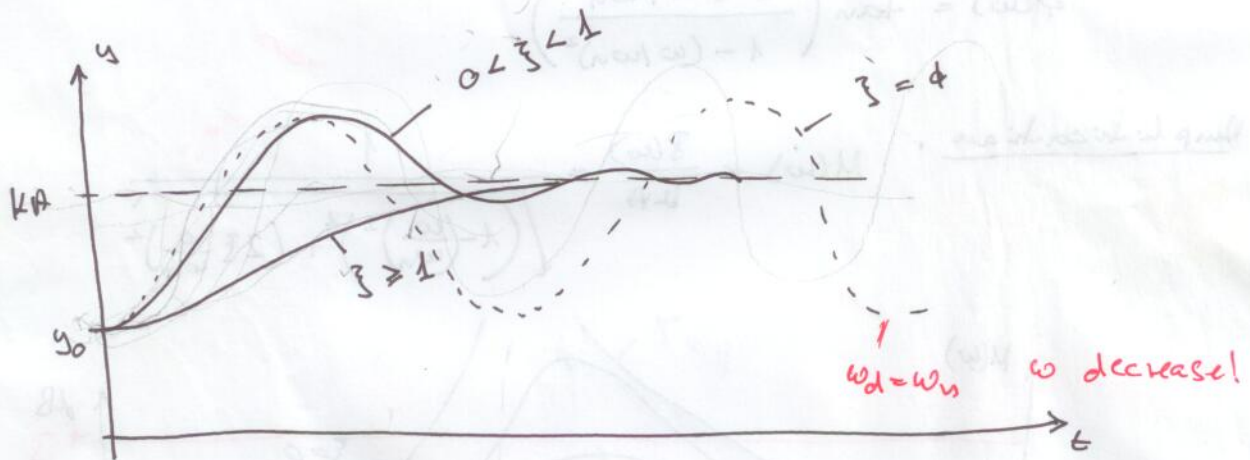
$y(t) = KA - KA \cdot e^{-\zeta \omega_n t} \left[ \frac{\zeta}{\sqrt{1-\zeta^2}} \sin(\omega_n \sqrt{1-\zeta^2} t) + \cos(\omega_n \sqrt{1-\zeta^2} t) \right]; 0 \leq \zeta < 1$

$y(t) = KA - KA (1 + \omega_n t) e^{-\omega_n t}; \zeta = 1$

$y(t) = KA - KA \left[ \frac{\zeta + \sqrt{\zeta^2 - 1}}{2\sqrt{\zeta^2 - 1}} e^{(-\zeta + \sqrt{\zeta^2 - 1}) \omega_n t} - \frac{\zeta - \sqrt{\zeta^2 - 1}}{2\sqrt{\zeta^2 - 1}} e^{(-\zeta - \sqrt{\zeta^2 - 1}) \omega_n t} \right]; \zeta > 1$

↓  
Steady state

transient response.



$T_d = \frac{2\pi}{\omega_d} = \frac{1}{f_d}$

$\omega_d = \omega_n \sqrt{1-\zeta^2}$

↳ ringing frequency /

damped natural frequency!

Good compromise:

$\zeta = 0.7$

Rise time:  $t_r$  • first reach 10% accuracy

In practice:

Settling time:  $t_s$  • stably reach 10% accuracy

$0.6 \leq \zeta \leq 0.8$

Sine function input

$$F(t) = A \cdot \sin(\omega t)$$

$$y(t) = y_n + \frac{KA \cdot \sin(\omega t + \phi(\omega))}{\sqrt{\left[1 - \left(\frac{\omega}{\omega_n}\right)^2\right]^2 + \left(2\zeta \frac{\omega}{\omega_n}\right)^2}} = \mathcal{B}(\omega) \sin(\omega t + \phi(\omega))$$

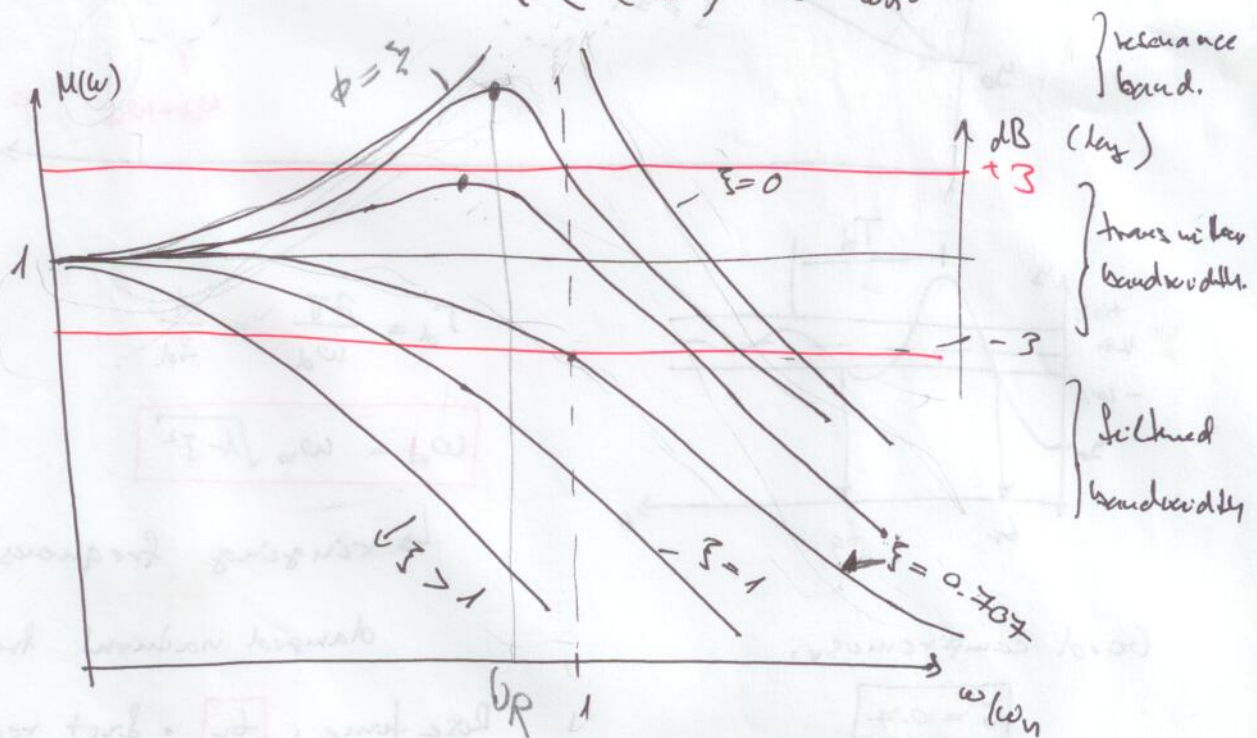
Steady state response!

$$\mathcal{B}(\omega) = \frac{KA}{\sqrt{\left(1 - \left(\frac{\omega}{\omega_n}\right)^2\right)^2 + \left(2\zeta \frac{\omega}{\omega_n}\right)^2}}$$

$$\phi(\omega) = \tan^{-1}\left(\frac{2\zeta \omega / \omega_n}{1 - (\omega / \omega_n)^2}\right)$$

Amplification

$$M(\omega) = \frac{\mathcal{B}(\omega)}{KA} = \frac{1}{\sqrt{\left(1 - \left(\frac{\omega}{\omega_n}\right)^2\right)^2 + \left(2\zeta \frac{\omega}{\omega_n}\right)^2}}$$



Peak: resonant frequency

$$\omega_R = \omega_n \sqrt{1 - 2\zeta^2}$$

i.e.  $\zeta = 0.707 \rightarrow$

$$\omega_R = \omega_n$$

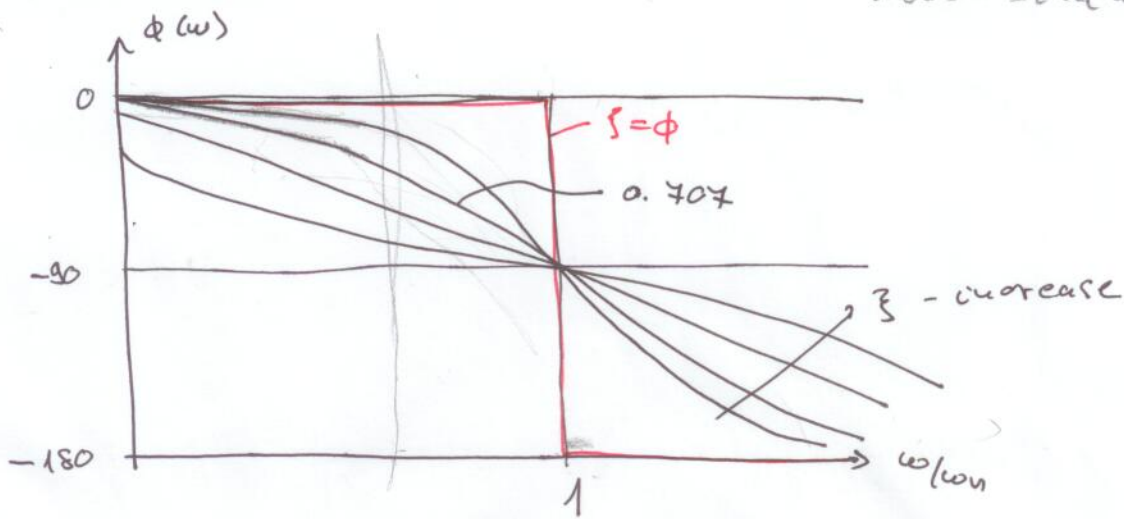
$$2\zeta^2 = 1$$

$$\zeta = \sqrt{0.5} = 0.7071$$



# phase linearity

$$u(t) = \sin(\omega_1 t) + \sin(\omega_2 t + \phi)$$



- there is always phase shift!
- it can be a problem:

## Example 9

• true signal  $u(t) = \sum_{n=1}^{\infty} \sin(n\omega t) = \sin(\omega t) + \sin(2\omega t) + \dots$

$\omega_n = n \cdot \omega$

- during the measurement there is a phase shift:

$$y(t) = \sin(\omega_1 t + \phi(\omega_1)) + \sin(2\omega_1 t + \phi(2\omega_1)) + \dots$$

- if  $\phi(\omega)$  is linear:

$$y(t) = \sin(\omega_1 t + \phi^*) + \sin(2\omega_1 t + 2\phi^*) + \dots = \sin(\omega_1 t + \phi) + \sin(2\omega_1 t + 2\phi) + \dots$$

$$\phi(\omega) = (\omega t + \phi)$$

$$y(t) = \sin \theta + \sin 2\theta + \dots \rightarrow \text{the signal waveform is the same!!}$$

there is no phase shift between the "true" and the measured signal!

# Fourier Ser (series)

Periodic function:  $f(t)$ ;  $f(t) = f(t+T)$

Fourier series:  $f(t) = a_0 + \sum_{i=1}^{\infty} a_i \cos(i \frac{2\pi}{T} t) + b_i \sin(i \frac{2\pi}{T} t)$

$$a_0 = \frac{1}{T} \int_0^T f(t) dt$$

$$\omega_i = i \cdot \frac{2\pi}{T}$$

$$a_i = \frac{2}{T} \int_0^T f(t) \cdot \cos(i \frac{2\pi}{T} t) dt$$

$$\Delta\omega = \frac{2\pi}{T}$$

$$b_i = \frac{2}{T} \int_0^T f(t) \cdot \sin(i \frac{2\pi}{T} t) dt$$

In practice:  $f_T(t) = a_0 + \sum_{i=1}^N a_i \cos(i \frac{2\pi}{T} t) + b_i \sin(i \frac{2\pi}{T} t)$

(truncated Fourier series)

How should  $a_0$ ,  $a_i$  and  $b_i$  be determined?

Orthogonal projection  
(vector analogy)

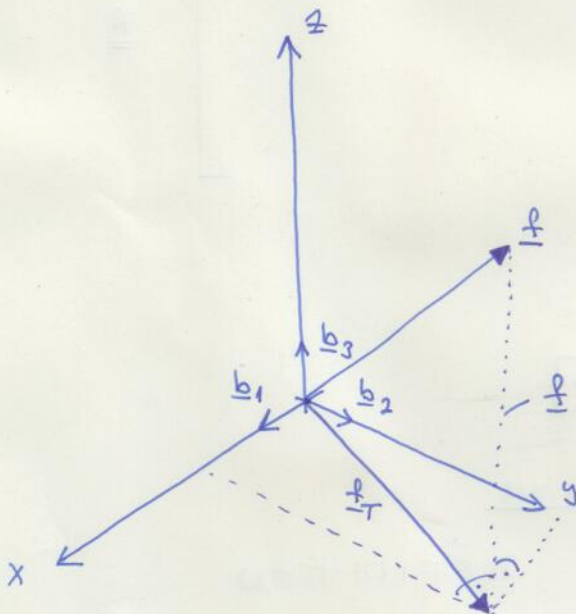
Basic vectors:  $\underline{b}_1, \underline{b}_2, \underline{b}_3$

Arbitrary vector: (in 3D)

$$\underline{f} = (a_1 \underline{b}_1 + a_2 \underline{b}_2 + a_3 \underline{b}_3)$$

Truncated vector: (in 2D)

$$\underline{f}_T = a_1 \underline{b}_1 + a_2 \underline{b}_2 \quad \begin{matrix} a_1 = ? \\ a_2 = ? \end{matrix}$$



Best approximation

$$\underline{f} - \underline{f}_T \perp \underline{b}_1 \quad \text{and}$$

$$\underline{f} - \underline{f}_T \perp \underline{b}_2$$

$$\left. \begin{aligned} \langle \underline{f} - \underline{f}_T, \underline{b}_1 \rangle &= 0 \\ \langle \underline{f} - \underline{f}_T, \underline{b}_2 \rangle &= 0 \end{aligned} \right\} \text{eq. sys.}$$

scalar product:  $\langle \cdot, \cdot \rangle$

$$\langle \underline{f} - a_1 \underline{b}_1 - a_2 \underline{b}_2, \underline{b}_1 \rangle = 0$$

$$\langle \underline{f} - a_1 \underline{b}_1 - a_2 \underline{b}_2, \underline{b}_2 \rangle = 0$$

$\underline{f}, \underline{b}_1, \underline{b}_2$  known vectors!

applying the rules of scalar product:

$$\langle \underline{f}, \underline{b}_1 \rangle - a_1 \langle \underline{b}_1, \underline{b}_1 \rangle - a_2 \langle \underline{b}_2, \underline{b}_1 \rangle = 0$$

$$\langle \underline{f}, \underline{b}_2 \rangle - a_1 \langle \underline{b}_1, \underline{b}_2 \rangle - a_2 \langle \underline{b}_2, \underline{b}_2 \rangle = 0$$

matrix system

$$\begin{bmatrix} \langle \underline{b}_1, \underline{b}_1 \rangle & \langle \underline{b}_2, \underline{b}_1 \rangle \\ \langle \underline{b}_1, \underline{b}_2 \rangle & \langle \underline{b}_2, \underline{b}_2 \rangle \end{bmatrix} \begin{bmatrix} a_1 \\ a_2 \end{bmatrix} = \begin{bmatrix} \langle \underline{f}, \underline{b}_1 \rangle \\ \langle \underline{f}, \underline{b}_2 \rangle \end{bmatrix}$$

$\underline{A} \quad \underline{x} \quad \underline{b}$

How apply to Fourier series?

Együtthatók:  $a_0, a_1, a_2, \dots, a_n, b_1, b_2, \dots, b_n$

Bázis:  $b_0 = 1; b_{ci} = \cos(i \frac{2\pi}{T} t); b_{si} = \sin(i \frac{2\pi}{T} t)$

$$\begin{bmatrix} \langle b_0, b_0 \rangle & \langle b_{c1}, b_0 \rangle & \dots & \langle b_{cn}, b_0 \rangle & \langle b_{s1}, b_0 \rangle & \dots & \langle b_{sn}, b_0 \rangle \\ \langle b_0, b_{c1} \rangle & \langle b_{c1}, b_{c1} \rangle & \dots & \langle b_{cn}, b_{c1} \rangle & & & \\ \vdots & & & & & & \\ \langle b_0, b_{sn} \rangle & & & & & & \end{bmatrix} \quad \underline{A}$$

$$\underline{x}^T = [a_0 \ a_1 \ \dots \ a_n \ b_1 \ \dots \ b_n]$$

$$\underline{b}^T = [\langle f, b_0 \rangle \ \langle f, b_{c1} \rangle \ \dots \ \langle f, b_{sn} \rangle]$$

scalar product in case of functions

$$\langle f, g \rangle = \int_0^T f(t) \cdot g(t) dt \quad \text{DEFINITION}$$

Fourier series has orthogonal basis:

$$\int_0^T \cos(i \frac{2\pi}{T} t) \cos(j \frac{2\pi}{T} t) dt = \langle b_{ci}, b_{cj} \rangle = \begin{cases} \phi & i \neq j \\ \frac{T}{2} & i = j \neq 1 \end{cases}$$

$$\int_0^T \sin(i \frac{2\pi}{T} t) \sin(j \frac{2\pi}{T} t) dt = \langle b_{si}, b_{sj} \rangle = \begin{cases} \phi & i \neq j \\ \frac{T}{2} & i = j \end{cases}$$

$$\int_0^T \sin(i \frac{2\pi}{T} t) \cos(j \frac{2\pi}{T} t) dt = \langle b_{si}, b_{cj} \rangle = \phi$$

$$\int_0^T 1 \cdot 1 dt = \langle b_0, b_0 \rangle = T$$

$$\begin{bmatrix} \langle b_0, b_0 \rangle \\ (T) \\ 0 \\ 0 \\ \vdots \\ 0 \end{bmatrix} \begin{bmatrix} 0 \\ \langle b_{c1}, b_{c1} \rangle \\ (T/2) \\ 0 \\ \langle b_{c2}, b_{c2} \rangle \\ (T/2) \\ \ddots \\ \langle b_{cn}, b_{cn} \rangle \\ (T/2) \end{bmatrix} = \begin{bmatrix} a_0 \\ a_1 \\ a_2 \\ \vdots \\ b_n \end{bmatrix} = \begin{bmatrix} \langle f, b_0 \rangle \\ \langle f, b_{c1} \rangle \\ \langle f, b_{c2} \rangle \\ \vdots \\ \langle f, b_{cn} \rangle \end{bmatrix}$$

$$a_0 = \frac{\langle f, b_0 \rangle}{\langle b_0, b_0 \rangle} = \frac{1}{T} \int_0^T f \cdot 1 dt$$

$$a_i = \frac{\langle f, b_{ci} \rangle}{\langle b_{ci}, b_{ci} \rangle} = \frac{2}{T} \int_0^T f \cdot b_{ci} dt = \frac{2}{T} \int_0^T f \cdot \cos(i \frac{2\pi}{T} t) dt$$

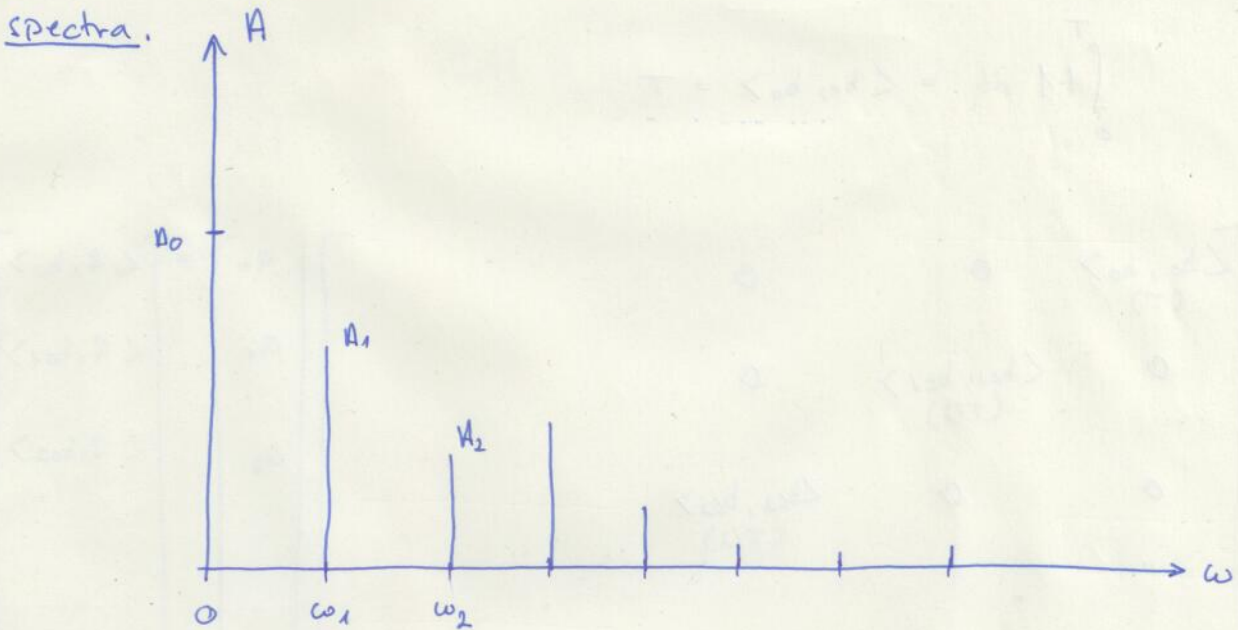
$$b_i = \frac{\langle f, b_{si} \rangle}{\langle b_{si}, b_{si} \rangle} = \frac{2}{T} \int_0^T f \cdot \sin(i \frac{2\pi}{T} t) dt$$

$$f_T(t) = a_0 + \sum_{i=1}^N a_i \cos(\omega_i t) + b_i \sin(\omega_i t)$$

$$\omega_i = i \frac{2\pi}{T} = i \omega$$

$$f_T(t) = a_0 + \sum_{i=1}^N A_i \cos(i\omega t + \phi)$$

$A_i = \sqrt{a_i^2 + b_i^2}$	amplitude
$\phi = \tan^{-1} \frac{b_i}{a_i}$	phase



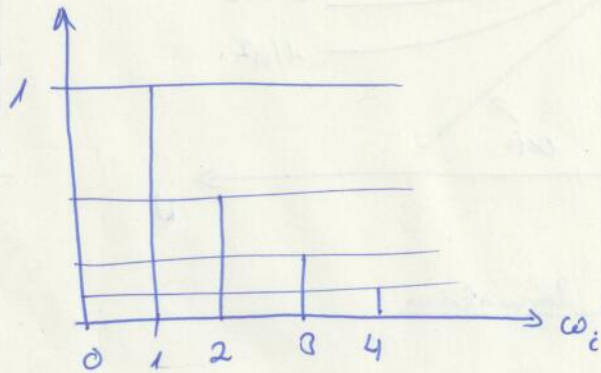
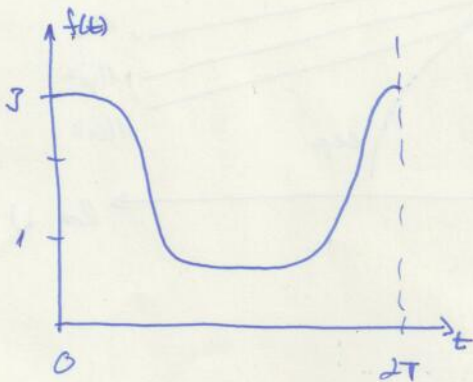
$$\omega = \Delta\omega = \frac{2\pi}{T}$$

$$\Delta\phi = \frac{1}{T}$$

Példák:  $f(t) = \frac{3}{5 - 4 \cdot \cos(x)}$ ;  $0 \leq t < T$ ;

①

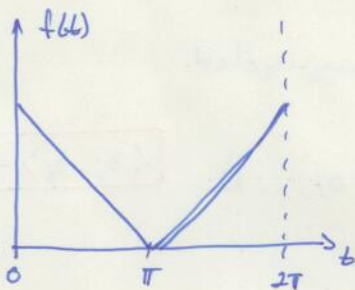
$C^\infty$  infinite many times differentiable (smooth function)



$A_i \approx 2 \cdot e^{-0.693n}$

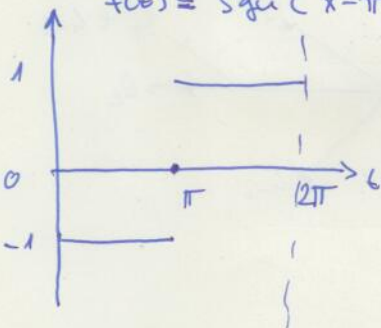
In general:  $a_i, b_i \sim O(\exp(-q \cdot n^r))$ ;  $q = 0.693$   
 $r = 1$   
 (exponential convergence)

②  $f(t) = \text{abs}(x - \pi)$



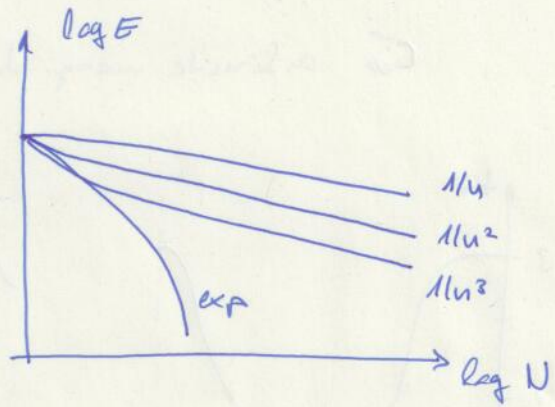
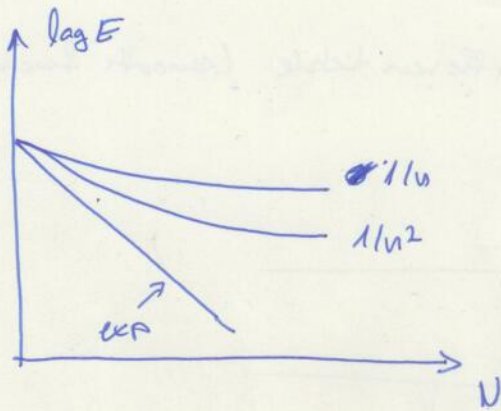
$a_i, b_i \sim O(1/n^2)$

③  $f(t) = \text{sgn}(x - \pi)$



$a_i, b_i \sim O(1/n)$

## Convergence:

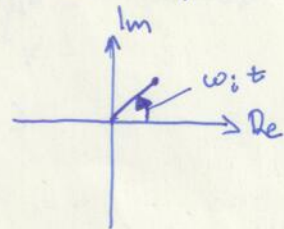


## Complex formalism

$f(t) = e^{j\omega t} = \cos \omega t + j \sin \omega t$   $j$  - complex unit vector

Let us write the basis as:  $b_i = e^{j\omega_i t}$  ;  $\omega_i = i \frac{2\pi}{T}$

$$f_T(t) = \sum_{i=-N}^N C_i b_i(t)$$

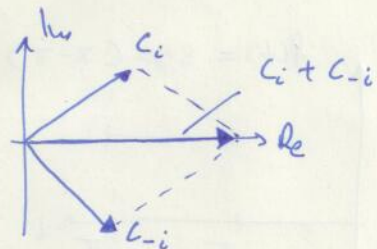


Inner product:  $\langle f, g \rangle = \int_0^T f \cdot \bar{g} dt$   
↑  
complex conjugated.

$$C_i = \frac{\langle f, b_i \rangle}{\langle b_i, b_i \rangle} \quad ; i = -N, \dots, 0, \dots, N \quad \langle b_i, b_i \rangle = T$$

If  $f(t)$  is real, then  $C_i$  are complex conjugate pairs.

$$C_i = \bar{C}_{-i}$$



The original coefficients

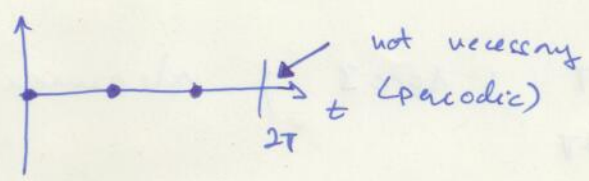
$$\begin{aligned} a_i &= C_i + C_{-i} \\ b_i &= j(C_i - C_{-i}) \end{aligned} \rightarrow \begin{matrix} A_i \\ \phi_i \end{matrix}$$

# Discrete Fourier Transform (DFT)

Total points:  $2N+1$  ; Time interval:  $0 \leq t \leq T$

$\Delta t = \frac{T}{2N+1}$  ;  $t_i = i \cdot \frac{T}{2N+1} = i \cdot \Delta t$   $i = 0 \dots 2N$

E.G.:  $N=1, i = 0, 1, 2$  ;  $\Delta T = \frac{2T}{3}$   $t_1 = 0$   
 $T=2T$   $t_2 = \frac{2T}{3}$   
 $t_3 = \frac{4T}{3}$



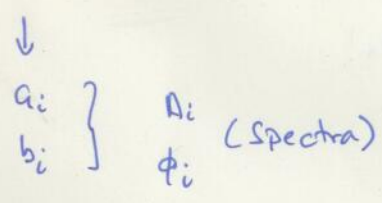
The function:  $f(t)$  continuous  
 $\underline{f} = f(t_i)$  discrete version, (sampled data)

Exponential basis:  $\underline{b}_k = e^{j\omega_k t_i}$  ;  $\omega_k = k \frac{2\pi}{T}$  ; complex valued vector (complex valued vector)

Inner product:  $\langle \underline{f}, \underline{g} \rangle = \sum_{i=0}^{2N} f(t_i) \cdot \bar{g}(t_i)$

$C_k = \frac{\langle \underline{f}, \underline{b}_k \rangle}{\langle \underline{b}_k, \underline{b}_k \rangle}$  ;  $\langle \underline{b}_k, \underline{b}_k \rangle = 2N+1$   
 (Matlab: do not make the division!)

$C_k = \bar{C}_{-k}$   $\neq$  complex conjugate.





a)  $f(t) = \frac{3}{5 - 4 \cdot \cos(4t)}$  ; smooth

b)  $f(t) = \text{abs}(-x + \pi)$  ; abs

c)  $f(t) = \text{sign}(x - \pi)$  ; discont ; discont long

d)  $f(t) = 3 \cdot \cos(x) + 5 \cdot \sin(5x)$  ; sepc (hint for deasy)

e)  $f(t) = 1 \cdot \sin(9x)$  ; spec2 ; explanation of deasy.

f)  $f(t) = \begin{cases} \sin(x), & x < 2\pi \\ \sin(5x), & x > 2\pi \end{cases}$  ; spec3 ; skaman 3-5

g)  $f(t) = (1 + 0.2 \sin(0.2x + 2)) \cdot \sin(3x)$  ; ampl-mod

h)  $f(t) = \sin\left(\left(3 + 0.2 \sin(0.2 \cdot x)\right) \cdot x\right)$  ; freq-mod-small

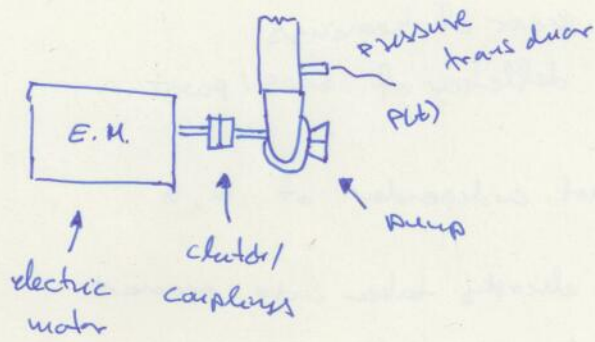
i) 0.2 ; freq-mod-large

j) chaotic referencia delbontas:  $\Delta f = \frac{1}{T}$   
(Mikor ellitod le!)

k) nonlinear chaotic, de egyes ban expl )  
E lötte a síma példá!

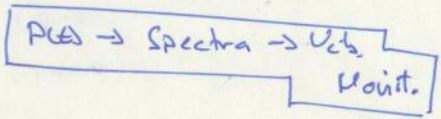
~~John Doe~~

# Vibration Monitoring



Goals: reveal peaks in the spectra and associate physical phenomena

- Hydrodynamic effects (pump)
- Mechanical effects (clutch, others like bearings)
- Motor instability

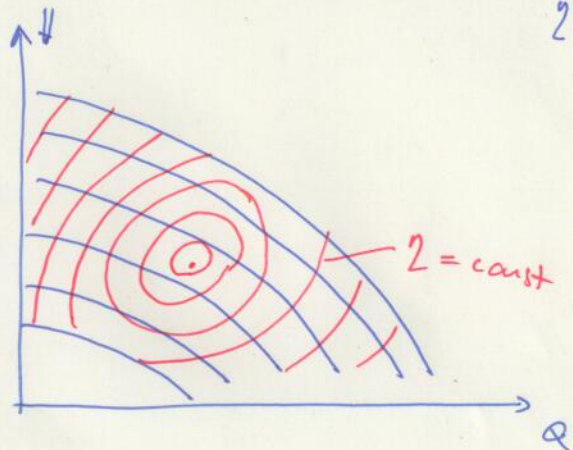
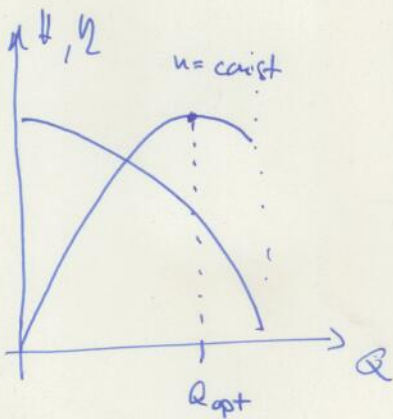


- Variable:  $P(t) \rightarrow$  Spectra

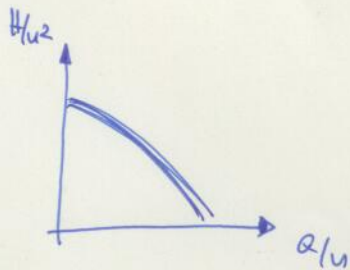
- Parameters:
- |                             |
|-----------------------------|
| H - head                    |
| Q - volume flow rate        |
| n - revolution number       |
| $P_i$ - input power of pump |

hydrod. parameters.  
characteristic curves  
control param. Q, n

$$\eta = \frac{Q \cdot \rho \cdot g \cdot H}{P_i}$$



+ Affinities:  
(Affinities)



every param. is controlled!

- Geometry of impeller / housing
- Number of blades

Mechanical parameters: eccentricity (geometric prop)  
 imbalance (mass direction of inertia)  
 alignment / fitting  
 wear of bearings  
 deflection of axis / pivot

Electric motor:  $\left. \begin{matrix} I \\ u \end{matrix} \right\}$  not independent of  $\Omega, n$   
 $n$  - already taken into account  
 stable / unstable behaviour

Equations: händersjället T  
 (temp)  
 output, electric noise

Strategy: water fall diagrams,  $f / f_r$ ;  $Q / Q_u$

A

B

$n = 630 \text{ 1/min}$  eccentr / imb  
 +  
 noise.

# Signal Processing

## • Static characteristics of a signal

### • Average or mean value:

$$\bar{y} = \frac{\int_{t_1}^{t_2} y(t) dt}{\int_{t_1}^{t_2} dt} = \frac{1}{t_2 - t_1} \cdot \int_{t_1}^{t_2} y(t) dt \quad \text{continuous}$$

$$\bar{y} = \frac{1}{N} \sum_{i=1}^N y_i \quad \text{discrete}$$

### • Root-mean-square RMS:

#### Example 1

Electrical power dissipation:  $P = I^2 R$

$$\text{Total: } \int_{t_1}^{t_2} P dt = \int_{t_1}^{t_2} I^2(t) \cdot R dt$$

$$\text{if } R \text{ is constant: } P_{\text{tot}} = R \int_{t_1}^{t_2} I^2 dt$$

$$\text{equivalent } I_e : P_{\text{tot}} = I_e^2 \cdot (t_2 - t_1) \cdot R \quad \left. \begin{array}{l} \\ \\ \end{array} \right\} I_e = \sqrt{\frac{1}{t_2 - t_1} \int_{t_1}^{t_2} I^2 dt} \quad \text{RMS}$$

$$y_{\text{rms}} = \sqrt{\frac{1}{t_2 - t_1} \int_{t_1}^{t_2} y^2(t) dt} \quad \text{continuous}$$

$$y_{\text{rms}} = \sqrt{\frac{1}{N} \sum_{i=1}^N y_i^2} \quad \text{discrete}$$

# Correlation

- Linear relationship between two data sets!

$$x_i = x_1, \dots, x_N \quad t_i = 1, \dots, N$$

$$y_i = y_1, \dots, y_N$$

$$R(x, y) = \frac{\sum_{i=1}^N (x_i - \bar{x})(y_i - \bar{y})}{S_x S_y}$$

$$S_x^2 = \sum_{i=1}^N (x_i - \bar{x})^2$$

$$S_y^2 = \sum_{i=1}^N (y_i - \bar{y})^2$$

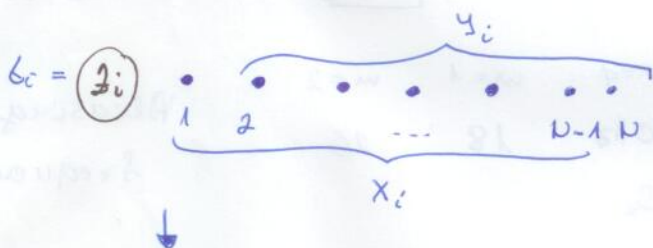
$$-1 \leq R \leq 1$$

$|R| \approx 1 \rightarrow$  Strong linear relationship

## Autocorrelation

$x_i$  } the set of data points  $t_i = 1, \dots, N$

Correlation with itself



$$x_i = z_i \quad i = 1, \dots, N-1$$

$$z_{i+1} = y_i \quad i = 1, \dots, N-1$$

First order auto correlation

$$R_1^A = \frac{\sum_{i=1}^{N-1} (x_i - \bar{x}_{(1)})(x_{i+1} - \bar{x}_{(2)})}{\sqrt{\sum_{i=1}^{N-1} (x_i - \bar{x}_{(1)})^2 \cdot \sum_{i=1}^{N-1} (x_{i+1} - \bar{x}_{(2)})^2}}$$

$$\bar{x}_{(1)} = \frac{1}{N-1} \sum_{i=1}^{N-1} x_i$$

$$\bar{x}_{(2)} = \frac{1}{N-1} \sum_{i=1}^{N-1} x_{i+1}$$

Simplifications

For long signals:  $\bar{x}_{(1)} \cong \bar{x}_{(2)} \cong \bar{x}$

$$\sum_{i=1}^{N-1} (x_i - \bar{x}_{(1)})^2 \cong \sum_{i=1}^{N-1} (x_{i+1} - \bar{x}_{(2)})^2 \cong$$

$$\sum_{i=1}^N (x_i - \bar{x})^2$$

2

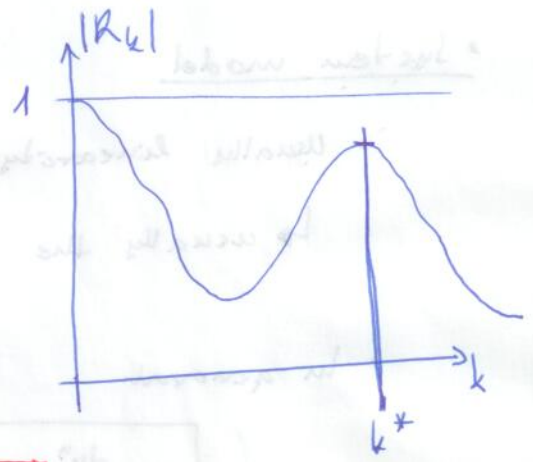
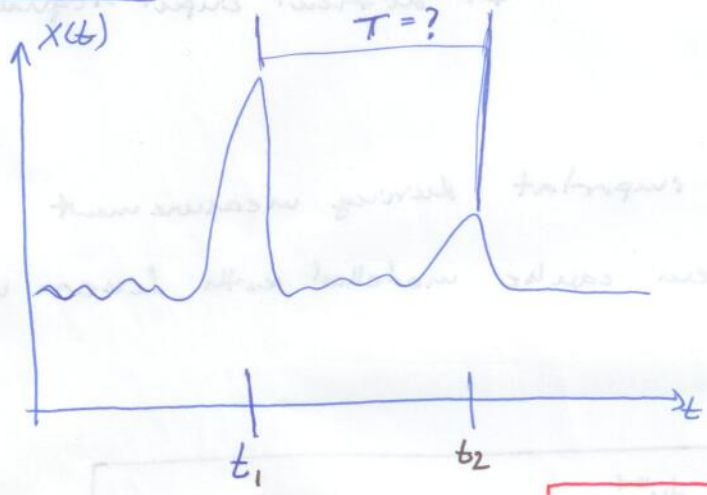
$$R_1^A(x) \cong \frac{\sum_{i=1}^{N-1} (x_i - \bar{x})(x_{i+1} - \bar{x})}{\sum_{i=1}^N (x_i - \bar{x})^2}$$

k - order autocorrelation:

$$R_k^A(x) \cong \frac{\sum_{i=1}^{N-k} (x_i - \bar{x})(x_{i+k} - \bar{x})}{\sum_{i=1}^N (x_i - \bar{x})^2}$$

shifted k times!

Application



$$t_2 - t_1 = T = k^* \cdot \Delta t$$

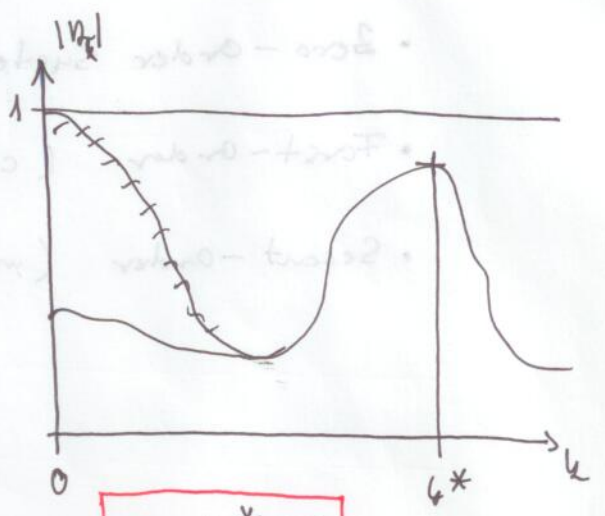
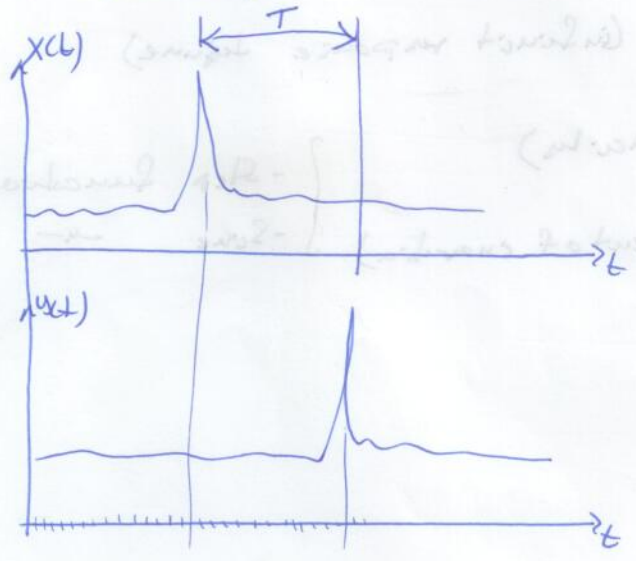
Cross-correlation

$x_i, y_i$  } distinct

$t_i = t_1, \dots, t_N$

usually:  $\Delta t = \text{const}$

$$R_k^C(x, y) \cong \frac{\sum_{i=1}^{N-k} (x_i - \bar{x})(y_{i+k} - \bar{y})}{\sqrt{\sum_{i=1}^N (x_i - \bar{x})^2 \cdot \sum_{i=1}^N (y_i - \bar{y})^2}}$$



$$T = k^* \cdot \Delta t$$